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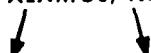
An Iterative Procedure for Alignment in
Underground Nuclear Testing

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An Iterative Procedure for Alignment in Underground Nuclear Testing

by

E. A. Kern



NOMENCLATURE

d_1	= outer beam diameter	r_0	= outer beam radius
d_2	= inner beam diameter	r_1	= radius of the large end of the frustum
\vec{G}	= vector from origin of xyz to origin of $x'y'z'$	r_2	= radius of the small end of the frustum
g	= magnitude of \vec{G}	s	= unit step function
h	= frustum height	\vec{S}	= vector directed along the support leg edge
J	= payoff function to be minimized	T	= orthogonal transformation matrix from xyz to XYZ system
K	= penalty constant	\vec{U}	= vector from origin of XYZ to origin of xyz
L	= length of support leg edge	\vec{V}	= vector from origin of xyz to a point on the frustum surface
\vec{O}	= vector from origin of XYZ to origin of $x'y'z'$	w	= width of a support leg
\vec{P}	= vector from origin of XYZ to end point of support leg edge	XYZ	= coordinate system with origin at the pipe center
\vec{Q}	= vector defining location of support leg edge relative to origin of $x'y'z'$	xyz	= coordinate system fixed to the frustum
\vec{R}	= vector from origin of XYZ to a point on the frustum surface	$x'y'z'$	= coordinate system fixed to the support plate
r_c	= perpendicular distance from the pipe centerline to an arbitrary point on the circumference of the small end of the frustum	z_p	= perpendicular distance from the large end of the frustum to an arbitrary cross-sectional plane
r_{cf}	= perpendicular distance from the pipe centerline to an arbitrary point on the frustum surface	β, γ, θ	= Euler rotation angles
r_{cfmin}	= minimum value of r_{cf}	ψ	= angle measured in a cross-sectional plane of the frustum
r_{cmax}	= maximum value of r_c	$[\]$	= column vector
r_f	= radius of an arbitrary frustum circular cross section	$()_x$	= x-coordinate of a vector
		$[\]_{XYZ}$	= vector in XYZ coordinates

AN ITERATIVE PROCEDURE FOR ALIGNMENT IN UNDERGROUND NUCLEAR TESTING

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ABSTRACT

A new method is presented for determining the mounting geometry for a frustum within the tunnel of an underground nuclear testing site. This method is based on a sequence of linear iterations in conjunction with a Davidon iterator for finding the minimum of a function of several parameters. Successful convergence of the method has been demonstrated on a time-sharing CDC 6600 computer. We believe this method could be generalized to other similar alignment problems.

I. INTRODUCTION

In the past, difficult geometric alignments associated with experiment packages in underground nuclear tests have been accomplished at the Los Alamos Scientific Laboratory by tedious drafting layouts. Recently, it was decided to attempt a computer solution of an extremely difficult alignment problem involving the placement of the frustum of a cone in the test tunnel. Early in the analysis it was concluded that an iterative-type approach would offer the best solution for this problem. Using a series of linear iterators in conjunction with a Davidon iterator, we successfully solved the problem with a CDC 6600 time-sharing computer system. This report describes the problem and presents the method of solution. To avoid security classification of this report, specific dimensions and computer results are not included.

II. DESCRIPTION OF THE PROBLEM GEOMETRY

A frustum of a cone is mounted in the test pipe as shown in Fig. 1. The beam diameters d_1 and d_2 depend

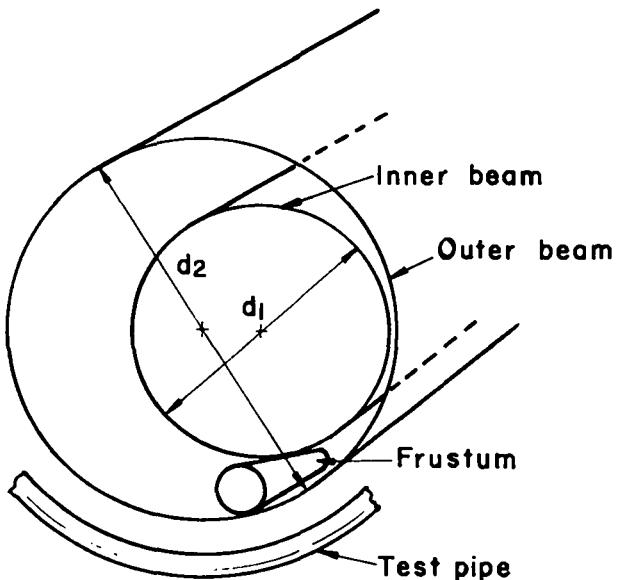


Fig. 1.
Location of frustum within a radiation pattern.

*W. C. Davidon, "Variable Metric Method for Minimization," AEC Research and Development report ANL-5990 (1959).

on the distance from the radiation source. The frustum is to remain in contact with the large or outer beam and is to be tangent at some point to the small or inner beam. This inner beam tangency requirement is imposed so that the frustum does not interfere with experiments farther down the test pipe. In addition, the centerline of the frustum is to be inclined at some given angle ϕ with the test pipe centerline. The frustum is mounted on a flat plate that, in turn, is supported on the wall of the test pipe by four support legs perpendicular to the mounting plate. The orientation and location of the frustum relative to the mounting plate and the location of the support legs relative to the mounting plate are fixed quantities (see Fig. 2). The basic problem is to determine the mounting locations and the dimensions of the four support legs on the test pipe so that the frustum satisfies the required alignment geometry.

III. ITERATIVE SOLUTION OF THE FRUSTUM ALIGNMENT PROBLEM

The first step in the solution is to determine how the frustum should be located and aligned relative to the inner and outer beams. For this purpose we define an XYZ-coordinate system with origin at the pipe center, with the Z-axis directed along the pipe centerline and the X-axis directed vertically upward (see Fig. 3). We also define an xyz-coordinate system fixed to the frustum with the x- and y-axes in the plane of the frustum base (large end of frustum) and the origin at the point where

the outer beam and the frustum base are tangent (see Fig. 3). The origins of the XYZ and xyz systems lie in the same cross-sectional plane of the pipe. The frustum is shown in Fig. 3 at the bottom of the pipe with the x-, y-, and z-axes respectively parallel to the X-, Y-, and Z-axes to simplify the geometry of the figure.

When the frustum is located and oriented correctly, the origin of the xyz system will not generally lie at the bottom of the pipe nor will the x-, y-, and z-axes be parallel to the X-, Y-, and Z-axes, respectively.

Because the xyz-coordinate system is fixed to the frustum, the orientation and position of the frustum will be determined by the origin and orientation of the xyz-coordinate system relative to the XYZ system. If the geometry of Fig. 3 is used as a starting point, the orientation and position of the xyz-coordinate system can be defined by a translation and three Euler angle rotations in the following manner.

(1) Translate the origin of the xyz system through the angle θ along the large beam circumference and simultaneously rotate the coordinate system about the z-axis so that the x-axis still points toward the center of the pipe (Fig. 4). In effect, the xyz system has been translated and rotated through an angle θ in the negative sense (right-hand rule) about the z-axis.

(2) Rotate the new xyz system in a negative sense about the x-axis through the angle β .

(3) Rotate the new xyz system in a positive sense about the y-axis through the angle γ .

This translation through the angle θ and the Euler rotation angles θ , β , and γ defines the position and orientation of the xyz system (hence the frustum) relative to

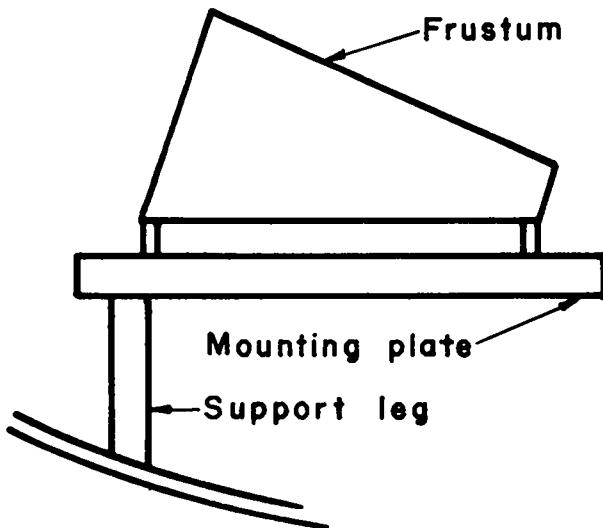


Fig. 2.
Frustum mounting geometry.

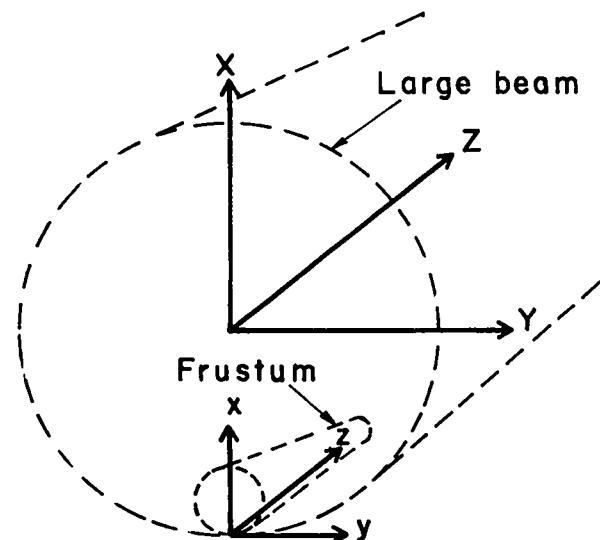


Fig. 3.
Definition of the XYZ- and xyz-coordinate systems.

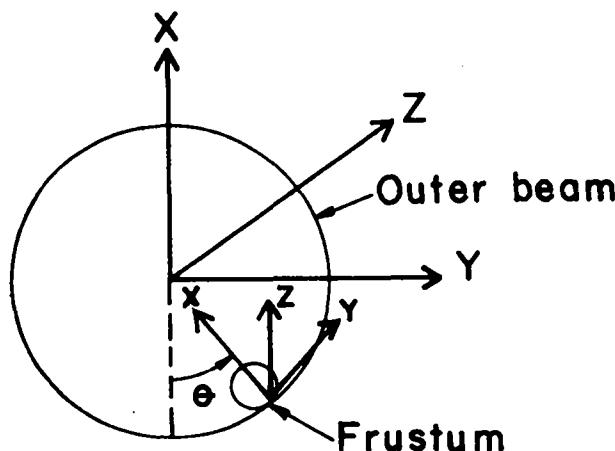


Fig. 4.

Displacement and rotation of the xyz system through the angle θ .

the XYZ system. Thus, the entire alignment procedure centers around the determination of the angles θ , β , and γ so that the frustum satisfies the specified requirements.

(1) The frustum base and the end of the frustum opposite the base are tangent to the large beam.

(2) The centerline of the frustum is inclined to the pipe centerline at some angle ϕ .

(3) The surface of the frustum is tangent at some point to the inner beam.

A direct method for finding the angles θ , β , and γ to satisfy the above three requirements did not immediately present itself; therefore, the following iterative approach was used.

(1) Estimate the pipe station at which the smaller end of the frustum (end opposite the base) is tangent to the outer beam and the pipe station at which the surface of the frustum is tangent to the inner beam. This establishes an estimate for the beam diameters at the corresponding tangency points.

(2) Guess the angles θ , β , and γ .

(3) Perform a linear iteration of the angle γ so that the end of the frustum opposite the frustum base is tangent to the outer beam.

(4) Perform a linear iteration on the angle β so that the centerline of the frustum is inclined by the desired angle ϕ to the pipe centerline. For each new angle β , return to step 3 and reiterate on the angle γ .

(5) Perform a linear iteration on the angle θ so that the surface of the frustum is tangent to the inner beam at some point. For each new angle θ , return to steps 3 and 4 to reiterate on the angles γ and β .

(6) Compute the actual pipe stations where the smaller end of the frustum is tangent to the outer beam and the surface of the frustum is tangent to the inner

beam. If the actual pipe stations differ by more than 0.0001 in. from the estimated pipe stations, set the estimated pipe stations equal to the actual pipe stations and return to step 2 of the iteration.

The above iteration procedure has proved successful in determining the angles θ , β , and γ and thus establishes the location and orientation of the frustum. The mathematical details associated with each of the above steps were purposely omitted so as not to obscure the basic iterative procedure.

The beam diameters along the pipe for both the outer and the inner beams increase as the distance down the pipe increases. Therefore, we must know the exact pipe stations at which the tangency points described in step 1 occur so that the outer and inner beam diameters can be established at these points. Because the pipe stations for the tangency points are not known initially, they must be estimated and then iterated. Note in step 6 that the iteration loop on the pipe stations is closed by setting the estimated pipe stations equal to the corresponding actual pipe stations at which the tangency points are computed to occur.

Very simple linear iterations are used in steps 3, 4, and 5 to determine the angles γ , β , and θ . Consider a linear iteration on the angle β where it is desired to obtain a specific inclination angle ϕ as an example. Let

$$d = \phi_{\text{desired}} - \phi_{\text{actual}},$$

$$d_n = \text{value of } d \text{ on the } n^{\text{th}} \text{ iteration},$$

and

$$\beta_n = \text{value of } \beta \text{ on the } n^{\text{th}} \text{ iteration.}$$

To compute β for the $(n+1)^{\text{th}}$ iteration, compute

$$\text{slope} = \frac{\beta_n - \beta_{n-1}}{d_n - d_{n-1}} . \quad (1)$$

Assume that this slope remains constant over the $(n+1)^{\text{th}}$ iteration, i.e.,

$$\text{slope} = \frac{\beta_{n+1} - \beta_n}{d_{n+1} - d_n} . \quad (2)$$

It is desired that d_{n+1} be zero on the $(n+1)^{\text{th}}$ iteration. Setting $d_{n+1} = 0$ in Eq. (2) and solving for β_{n+1} gives

$$\beta_{n+1} = \beta_n - \text{slope} (d_n) . \quad (3)$$

Iteration for β continues until the absolute value of d becomes less than a prespecified tolerance. The linear iterations on γ and θ are performed in the same manner.

In step 3 of the iteration, we must determine the angle γ so that the small end of the frustum (end opposite the base) is tangent to the outer beam circumference. This is accomplished by defining a vector \vec{R} , which locates a point on the circumference of the small end of the frustum relative to the center of the pipe (see Fig. 5). Let the X-, Y-, and Z-components of \vec{R} be denoted by R_x, R_y , and R_z , respectively, and let r_c be the distance of a given point on the small end of the frustum from the pipe center. Then it follows that

$$r_c = (R_x^2 + R_y^2)^{\frac{1}{2}} . \quad (4)$$

The value of r_c will vary as \vec{R} moves around the small end of the frustum. The maximum value of r_c , r_{cmax} , defines the point on the small end of the frustum that is farthest from the center of the pipe. If r_{cmax} is smaller than the outer beam radius, the small end of the frustum lies inside the outer beam, whereas if r_{cmax} is larger than the outer beam radius, a portion of the small end of the frustum lies outside the outer beam. For the small end of the frustum to be tangent to the outer beam, the r_{cmax} must be equal to the outer beam radius. Linear iteration on γ is therefore performed until r_{cmax} is within 10^{-8} in. of the outer beam radius.

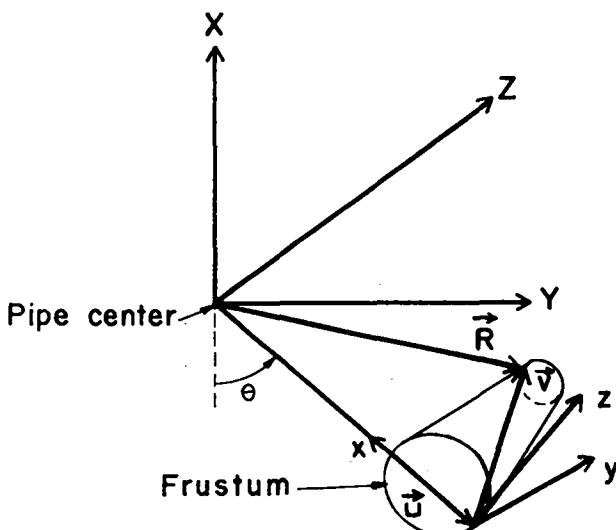


Fig. 5.

Vector \vec{R} from pipe center to point on circumference of small end of frustum.

We compute r_{cmax} for a given angle γ as follows. From Fig. 5 we have

$$\vec{R} = \vec{U} + \vec{V} , \quad (5)$$

where

\vec{U} = the vector from the origin of the XYZ system to the origin of the xyz system

and

\vec{V} = the vector from the origin of the xyz system to a point on the small end of the frustum.

The vector \vec{U} is easily established because the origin of the xyz system lies in the XY plane. Referring to Fig. 5 and letting r_0 be the radius of the outer beam, we note that

$$U_x = -r_0 \cos \theta \quad (6)$$

and

$$U_y = r_0 \sin \theta . \quad (7)$$

The vector \vec{V} is most easily determined in the xyz system and is then transformed into the XYZ system. Here we consider Fig. 6 where the vector \vec{V} is shown equivalent to the sum of the three vectors, \vec{V}_1 , \vec{V}_2 , and \vec{V}_3 , and the

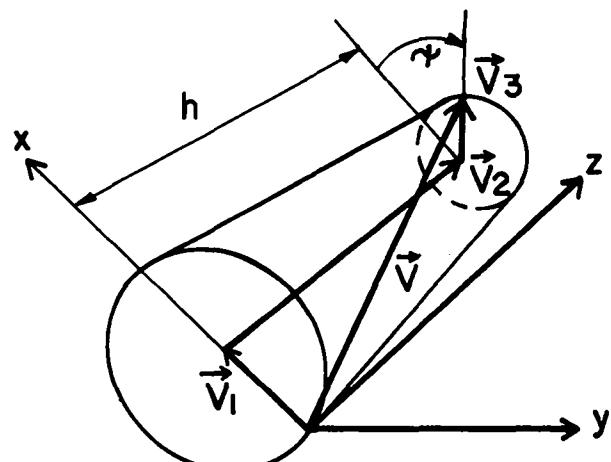


Fig. 6.

The vector \vec{V} locating an arbitrary point on the small end of the frustum relative to the xyz system.

radii of the large and small ends of the frustum are given by r_1 and r_2 , respectively. The angle ψ is measured in the plane defined by the small end of the frustum and is measured from a line parallel to the x-axis to the vector \vec{V}_3 . The frustum height is denoted by h . Thus, the vectors \vec{V}_1 , \vec{V}_2 , and \vec{V}_3 are as follows in the xyz system:

$$\vec{V}_1 = \begin{bmatrix} r_1 \\ 0 \\ 0 \end{bmatrix} : \vec{V}_2 = \begin{bmatrix} 0 \\ 0 \\ h \end{bmatrix} : \vec{V}_3 = \begin{bmatrix} r_2 \cos \psi \\ r_2 \sin \psi \\ 0 \end{bmatrix} . \quad (8)$$

Because

$$\vec{V} = \vec{V}_1 + \vec{V}_2 + \vec{V}_3$$

it follows that

$$[\vec{V}]_{xyz} = \begin{bmatrix} r_1 + r_2 \cos \psi \\ r_2 \sin \psi \\ h \end{bmatrix} . \quad (9)$$

To find the vector \vec{R} we must transform the coordinates of the vector \vec{V} into the XYZ system. The orientation of the xyz system relative to the XYZ system is defined by the Euler angles θ , β , and γ . This then defines an orthogonal transformation matrix T that transforms the coordinates of the vector \vec{V} from the xyz system to the XYZ system, i.e.,

$$[\vec{V}]_{XYZ} = T [\vec{V}]_{xyz} \quad (10)$$

where

$$T = \begin{bmatrix} p_1 & p_4 & p_7 \\ p_2 & p_5 & p_8 \\ p_3 & p_6 & p_9 \end{bmatrix}$$

and

$$\begin{aligned} p_1 &= \cos \theta \cos \gamma - \sin \theta \sin \beta \sin \gamma, \\ p_2 &= -\sin \theta \cos \gamma - \sin \beta \sin \gamma \cos \theta, \\ p_3 &= -\sin \gamma \cos \beta, \\ p_4 &= \cos \beta \sin \theta, \\ p_5 &= \cos \theta \cos \beta, \\ p_6 &= -\sin \beta, \\ p_7 &= \cos \theta \sin \gamma + \sin \theta \sin \beta \cos \gamma, \\ p_8 &= -\sin \theta \sin \gamma + \cos \theta \sin \beta \cos \gamma, \\ p_9 &= \cos \gamma \cos \beta. \end{aligned}$$

and

$$p_9 = \cos \gamma \cos \beta.$$

Substituting Eqs. (6), (7), (9), and (10) into Eq. (5) gives for the vector \vec{R} in XYZ coordinates

$$[\vec{R}]_{XYZ} = r_0 \begin{bmatrix} -\cos \theta \\ \sin \theta \\ 0 \end{bmatrix} + T \begin{bmatrix} r_1 + r_2 \cos \psi \\ r_2 \sin \psi \\ h \end{bmatrix} . \quad (11)$$

The X- and Y-components of the vector \vec{R} can then be used in Eq. (4) to solve for r_c .

The point on the small end of the frustum that yields r_{cmax} is found by a one-dimensional search on the parameter ψ . Iteration on the Euler angle γ is complete when r_{cmax} is equal to the outer beam radius. This is equivalent to the small end of the frustum being tangent to the circumference of the outer beam.

In step 4 of the iteration, angle β is set to yield the desired angle ϕ between the pipe and the frustum axes. Now the frustum axis is parallel to the z-axis. Thus the cosine of the angle between the pipe and the frustum axes will be equal to the dot product of the unit vectors directed along the Z- and z-axes. This is equivalent to element p_9 of the matrix T . Hence,

$$\phi = \cos^{-1} (\cos \gamma \cos \beta) . \quad (12)$$

Linear iteration is performed on the angle β until the actual inclination angle ϕ is equal to within 10^{-6} rad of the desired inclination angle.

With γ and β determined, the angle θ is computed by a linear iteration so that the surface of the frustum is tangent to the inner beam. This tangency point is determined in a manner similar to that used for determining the tangency point at the outer beam and small end of the

frustum. Here, however, the tangency point can lie at any point on the frustum surface. Let \vec{R} be the vector directed from the origin of the XYZ system to any point on the frustum surface. The geometry of Fig. 5 is applicable if the vector \vec{R} is not restricted to the small end of the frustum. Let \vec{U} be the vector joining the XYZ and xyz systems, and let \vec{V} be the vector from the origin of the xyz system to the end point of the vector \vec{R} . Equations (5)-(7) are applicable here. The vector \vec{V} can be expressed in terms of xyz coordinates by a consideration of the frustum shown in Fig. 7. Again \vec{V} is equal to the sum of the vectors \vec{V}_1 , \vec{V}_2 , and \vec{V}_3 . The vector \vec{V}_1 is as defined in Eq. (8). The vector \vec{V}_2 is still directed along the z -axis but it is now of variable length z_p , i.e.,

$$[\vec{V}_2]_{xyz} = \begin{bmatrix} 0 \\ 0 \\ z_p \end{bmatrix} . \quad (13)$$

The vector \vec{V}_3 lies in a plane parallel to the xy plane and is directed from the frustum centerline to the frustum surface. Let r_f be the radius of the frustum at a cross section lying at a distance z_p from the frustum base. Then

$$r_f = r_1 - \frac{z_p (r_1 - r_2)}{h} , \quad (14)$$

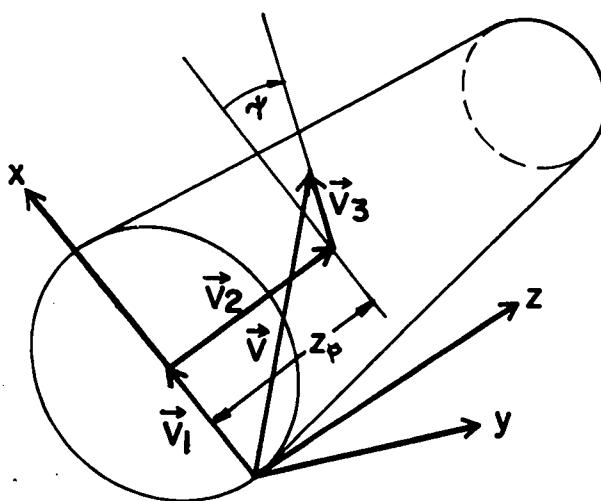


Fig. 7.
The vector \vec{V} locating an arbitrary point on the frustum surface relative to the xyz system.

where

r_1 = the radius of the frustum base (large end),
 r_2 = the radius of the small end of the frustum,

and

h = the height of the frustum.

With ψ defined as the angle in a plane parallel to the xy plane and measured between a line parallel to the x -axis and the vector \vec{V}_3 , it follows that

$$[\vec{V}_3]_{xyz} = \begin{bmatrix} r_f \cos \psi \\ r_f \sin \psi \\ 0 \end{bmatrix} . \quad (15)$$

Summing the vectors \vec{V}_1 from Eq. (8), \vec{V}_2 from Eq. (13), and \vec{V}_3 from Eqs. (14) and (15) gives for the vector \vec{V}

$$[\vec{V}]_{xyz} = \begin{bmatrix} r_1 + \left[r_1 - \frac{z_p (r_1 - r_2)}{h} \right] \cos \psi \\ r_1 - z_p \frac{(r_1 - r_2)}{h} \sin \psi \\ z_p \end{bmatrix} . \quad (16)$$

The vector \vec{R} can now be obtained by transforming the vector \vec{V} by way of the matrix T and adding the result to the vector \vec{U} defined by Eqs. (5) and (6), i.e.,

$$[\vec{R}]_{XYZ} = r_o \begin{bmatrix} -\cos \theta \\ \sin \theta \\ 0 \end{bmatrix} + T \begin{bmatrix} r_1 + \left[r_1 - z_p \frac{(r_1 - r_2)}{h} \right] \cos \psi \\ r_1 - z_p \frac{(r_1 - r_2)}{h} \sin \psi \\ z_p \end{bmatrix} . \quad (17)$$

As seen from Eq. (17), for fixed angles θ , β , and γ , the vector \vec{R} is a function of the distance z_p and the angle ψ . We now find that point on the frustum surface that lies closest to the pipe centerline. With

r_{cf} = to the perpendicular distance from the pipe centerline to the frustum surface

it follows that

$$r_{cf} = (R_x^2 + R_y^2)^{1/2} . \quad (18)$$

The minimum value of r_{cf} , r_{cfmin} , is the minimum distance between the pipe centerline and the frustum surface. Thus, if r_{cfmin} is less than the inner beam radius, a portion of the frustum surface lies inside the inner beam, whereas if r_{cfmin} is greater than the inner beam radius, there is no common point between the frustum and the inner beam. When r_{cfmin} is equal to the inner beam radius, the inner beam and the frustum surface are tangent. Therefore, we must compute r_{cfmin} .

For fixed angles θ , β , and γ the distances R_x and R_y are functions of z_p and ψ . Hence, r_{cfmin} can be found by minimizing r_{cf} with respect to the parameters z_p and ψ subject to the constraint that the minimum point actually lies upon the frustum. Let J be the function of z_p and ψ to be minimized. Then

$$J = (R_x^2 + R_y^2)^{1/2} + K \left[s(z_p - h) (z_p - h)^2 + s(z_p) z_p^2 \right] , \quad (19)$$

where

K = large penalty constant

and

$$s(f) = \text{step function} = \begin{cases} 1 & f > 0 \\ 0 & f < 0 \end{cases} .$$

Note from Eq. (19) that the function J is composed of the quantity to be minimized ($r_{cf} = \sqrt{R_x^2 + R_y^2}$) and a penalty term to ensure that the minimum does not lie at some value of z_p , which is not on the frustum surface. The values of z_p and ψ , which minimize the function J and consequently r_{cf} , are obtained with the aid of a Davidon iterator. The Davidon iterator is a second-order method for finding the minimum of a multiparameter function. Rapid and reliable convergence to the minimum

can be obtained with this iterator for the type function J defined by Eq. (19). With the minimum of r_{cf} computed in this way, linear iteration on the angle θ is executed until the difference between r_{cfmin} and the inner beam radius is less than 10^{-6} in.

IV. COMPUTATION OF THE FRUSTUM SUPPORT GEOMETRY

Exact determination of the frustum location and orientation has been outlined above. As shown in Fig. 2, the frustum is mounted on a flat plate that, in turn, is supported by four support legs welded to the pipe wall. The location and orientation of the frustum relative to the support plate and the location of the legs relative to the support plate are prespecified quantities. We must determine the dimensions of the support legs and the points at which the support legs should be welded to the pipe.

A typical support leg is shown in Fig. 8. The top of the support leg is fastened perpendicular to the support plate while the bottom surface of the support leg rests on the inside surface of the test pipe. All cross-sectional planes of the support leg that are parallel to the top are squares of width w . We must find the lengths l_1 , l_2 , l_3 , and l_4 of the support leg edges so that the support legs will rest firmly on the pipe surface and at the same time provide the proper location and orientation of the frustum. In addition, the points at which these edges intersect the

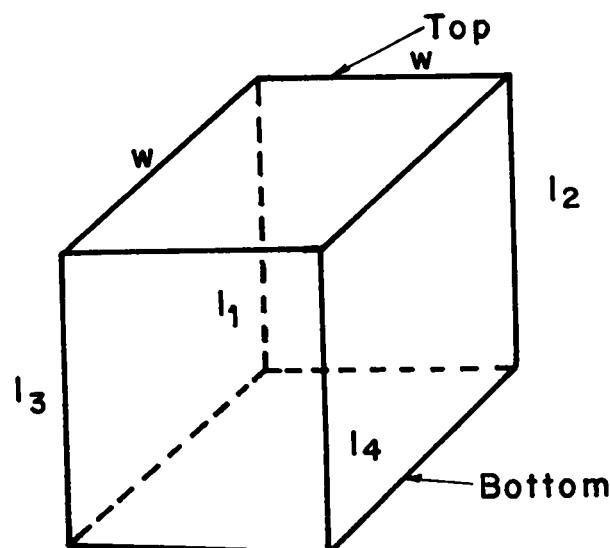


Fig. 8.
Support leg geometry.

pipe are needed to locate the support plate properly within the pipe.

A support leg edge, along with the support plate and pipe, is shown in Fig. 9. The XYZ-coordinate system, with origin at the pipe center and the Z-axis directed along the pipe centerline, is identical to the XYZ system previously defined. The $x'y'z'$ -coordinate system shown in Fig. 9 is fixed to the support plate with the origin of the system lying in the center of the plate bottom. The x' -axis is normal to the plate surface and the z' -axis is directed along the centerline of the bottom surface. The origins of the XYZ and $x'y'z'$ systems lie in the same cross-sectional plane of the pipe. The $x'y'z'$ system can be derived from the xyz system by translating the origin of the xyz system from the bottom of the frustum to the bottom of the support plate and by rotating this translated system about the y-axis through some angle η so that the z' -axis lies in the bottom surface of the plate. Thus, the Euler angles that define the orientation of the $x'y'z'$ system relative to

the XYZ system are θ , β , and γ' , where θ and β are the same as defined in Sec. III and

$$\gamma' = \gamma + \eta \quad . \quad (20)$$

Let T' be the matrix that transforms the coordinates of a vector from the $x'y'z'$ system to the XYZ system. Then T' can be obtained from the matrix T defined in Sec. III by replacing γ with γ' .

The vector \vec{O} shown in Fig. 9 locates the origin of the $x'y'z'$ system relative to the XYZ system, and the sum of the vectors \vec{Q} and \vec{S} locates the extremity of the support leg edge relative to the origin of the $x'y'z'$ system. The vector \vec{P} locates the extremity of the edge relative to the XYZ system. Therefore, the point at which the edge intersects the pipe is defined by the coordinates of \vec{P} , and the length of the edge is equal to the length of \vec{S} .

We must determine the length of the vector \vec{S} so that the end point of \vec{S} touches the inner wall of the pipe. This is equivalent to requiring that the radial distance of the end point of \vec{S} from the center of the pipe be equal to the pipe radius. Let P_x and P_y be the X- and Y-coordinates, respectively, of \vec{P} . Then the radial distance from the center of the pipe will equal $(P_x^2 + P_y^2)^{1/2}$. Hence, we must determine the length of the vector \vec{S} so that

$$\text{pipe radius} = (P_x^2 + P_y^2)^{1/2} \quad . \quad (21)$$

From Fig. 9 we know that \vec{P} is the sum of the vectors \vec{O} , \vec{Q} , and \vec{S} , i.e.,

$$\vec{P} = \vec{O} + \vec{Q} + \vec{S} \quad . \quad (22)$$

The vector \vec{O} can be represented as the sum of the vectors \vec{U} and \vec{G} (see Fig. 10), where the vector \vec{U} is directed from the origin of XYZ to the origin of xyz, and the vector \vec{G} is directed from the origin of xyz to the origin of $x'y'z'$. Both vectors \vec{U} and \vec{G} lie in the XY plane with the X- and Y-coordinates of \vec{U} given by Eqs. (6) and (7). The vector \vec{G} has a specified length equal to g and is directed along the negative x' -axis, i.e.,

$$[\vec{g}]_{x'y'z'} = \begin{bmatrix} -g \\ 0 \\ 0 \end{bmatrix} \quad . \quad (23)$$

Fig. 9.

Vectors \vec{O} , \vec{P} , \vec{Q} , and \vec{S} defining the extremities of a support leg edge relative to the support plate and the pipe center.

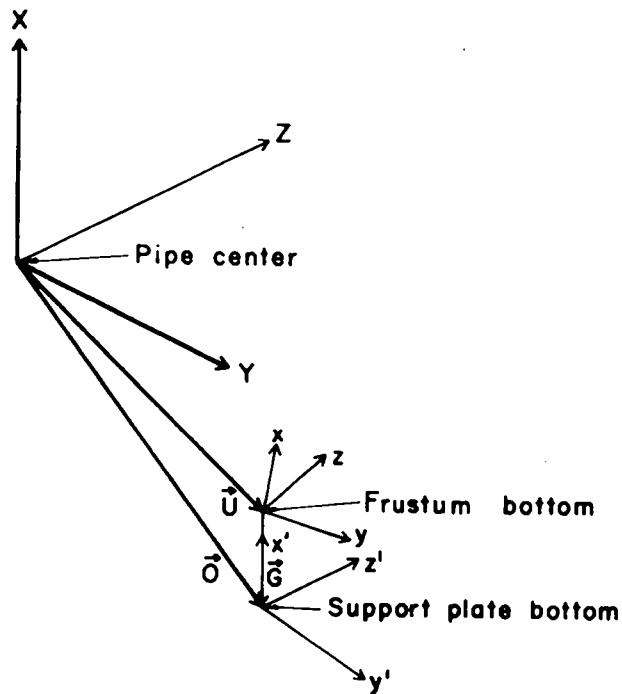


Fig. 10.
Representation of the vector \vec{O} as the vector sum of \vec{U} and \vec{G} .

Transforming the vector \vec{G} into the XYZ system by means of the matrix T' and forming the vector \vec{O} as the vector sum of \vec{U} and \vec{G} gives

$$[\vec{o}]_{XYZ} = \begin{bmatrix} -r_0 \cos \theta \\ r_0 \sin \theta \\ 0 \end{bmatrix} + T' \begin{bmatrix} -g \\ 0 \\ 0 \end{bmatrix}. \quad (24)$$

The vector \vec{Q} shown in Fig. 9 provides the location of the support edge relative to the plate. Hence, the y' - and z' -coordinates of Q , Q_y , and Q_z are readily obtainable as input to the alignment problem. Obviously the x' -component of Q will be zero because Q lies in the bottom of the support plane and x' is normal to this plane, i.e.,

$$[\vec{Q}]_{x'y'z'} = \begin{bmatrix} 0 \\ Q_y \\ Q_z \end{bmatrix}, \quad (25)$$

where Q_y and Q_z are known values.

The vector \vec{S} has a component only in the negative x' direction because this vector is normal to the support plate and therefore normal to the $y'z'$ plane. Hence, \vec{S} can be represented in the $x'y'z'$ system as follows:

$$[\vec{S}]_{x'y'z'} = \begin{bmatrix} -L \\ 0 \\ 0 \end{bmatrix}, \quad (26)$$

where L equals the unknown length of the vector \vec{S} .

Transforming \vec{Q} and \vec{S} from Eqs. (25) and (26), respectively, by means of the matrix T' and substituting these transformed vectors along with the vector \vec{O} from Eq. (24) into Eq. (22) gives for the vector \vec{P}

$$[\vec{P}]_{XYZ} = \begin{bmatrix} -r_0 \cos \theta \\ r_0 \sin \theta \\ 0 \end{bmatrix} + T' \begin{bmatrix} -g-L \\ Q_y \\ Q_z \end{bmatrix}. \quad (27)$$

With the expression for \vec{P} as above, a linear iteration on the length L is carried out until Eq. (21) is satisfied to within 10^{-6} in. This yields the desired length of the support leg edge. Also, the coordinates of the vector \vec{P} define the point where the support leg edge touches the inside of the test pipe.

V. CONCLUSIONS

This iteration technique has worked successfully in solving the problem of aligning the frustum of a cone in the test tunnel of an underground nuclear testing site. Thus, the feasibility of using an iterative approach in conjunction with a high-speed digital computer to handle complex geometric alignment problems of this nature has been proved. Rapid convergence of the iterator was experienced with the total computation time on the CDC 6600 computer being less than 3 sec.

Single precision arithmetic was used throughout the computation to conserve computer core storage on the time-sharing system. For the CDC computer this provides

about 14 decimal digits of accuracy. The Davidon iterator had to be "tuned" to operate within these accuracy limitations because 14 decimal digits are only marginally sufficient to ensure reliable convergence of this type of iterator. Double precision arithmetic, which essentially doubles the number of accurate decimal digits, would have eliminated these difficulties.

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